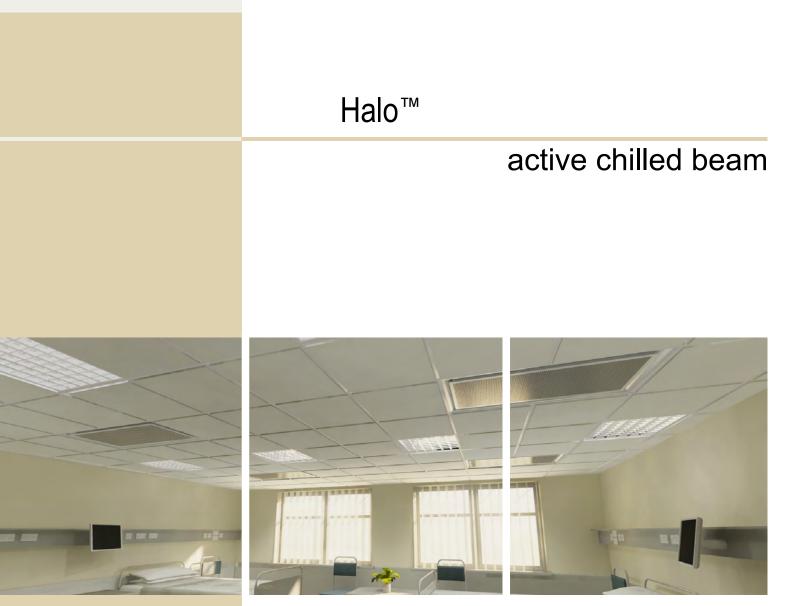
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Product Description Construction **Cooling Performance** Heating Performance **Cooling Selection Tables** Heating Selection Tables Air Cooling Effect Scatter Diagram **Product Dimensions Mounting Details Perforation Pattern Options** Product Ordering Codes **Calculation Program** Project Specific Testing Facility Photometric Testing Facility Acoustic Testing Facility

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Product Description

Halo is one of the FTF Group's latest range of high performance Chilled Beams. Energy efficiency has been a key driver for such advancements in the FTF Group's Chilled Beam Technology.

Halo is only ⁹.1¹¹ deep and can achieve up to ⁵¹⁵⁸ BTU/hr total cooling (based on a ⁴ft long beam with a ¹⁸ dTF between room and mean water temperature and ⁹⁴ CFM of air ^{60°}F with a ⁰.⁴inH₂^o).

The Halo beam contains a number of **Patented performance enhancing features** and Registered Designs for aesthetic enhancements, all as can be expected from the FTF Group's brand.

These high-capacity active chilled beams have a small footprint and as such have become increasingly popular as they can free up ceiling area whilst still handling significant heat gains and heat losses. However, the challenge has been to meet these demands whilst still delivering high levels of occupancy comfort. The FTF Group's Halo active chilled beam meets these challenges with its unique, true ^{360°} air discharge characteristic with concealed air discharge veins.

The latest-generation of ^{360°} Active Chilled Beam combines cooling and optional heating function with a revolutionary air discharge system and pattern. By introducing the air with set back air deflector veins further up into the point of discharge rather than being mounted on the underplates like earlier models, this not only improve the ^{360°} diffusion pattern it also vastly improves the products aesthetics. This latest development is a Registered Design in addition to the Patented performance enhancing items by the FTF Group. When compared to traditional ²-way or ⁴-way discharge pattern by others, Halo can deliver a reduction in air velocities of up to ³⁵%.

This optimal method of spreading the air in all directions means the shortest possible air throws are created, resulting in optimal levels of comfort to building occupants.



Halo is also available with a **drop down heat exchange battery** for easy cleaning to all ⁴ sides of the heat exchanger - contact FTF Group's technical department for further information.

At a glance

- Halo is only ⁹.1" deep and can achieve up to ⁵¹⁵⁸ BTU/hr total cooling.
- High-capacity active chilled beams with a small footprint.
- True ^{360°} air discharge characteristic.
- Concealed air discharge veins.
- Spreading the air in all directions means the shortest possible air throws are created.
- Halo is offered in ³ standard models; "I", "C" and "F":
 - Halo "I" models are for integrated ceiling installation.
 - Halo-"C"-⁶⁰ and Halo-"C"-¹²⁰ are designed for integration into metal clip-in ceiling systems.
 - Halo "F"-⁶⁰ is designed for free-hanging exposed applications.
- Providing a comfortable environment, compliant to BS EN ISO ⁷⁷³⁰/ASHRAE ⁵⁵.

Construction

Halo is offered in ³ standard models; "I", "C" and "F".

Halo "I" models are for integrated ceiling installation in standard ⁰.⁶" or ⁰.⁹" exposed tee bar grids (Lay-In grid systems) replacing ²³.⁶" x ²³.⁶" or ⁴⁷.²" x ²³.⁶" tile modules and can be used for integration with either "mineral fiber" tiles or plaster board ceilings.

Halo-"C"-⁶⁰ and Halo-"C"-¹²⁰ are designed for integration into metal clip-in ceiling systems.

Halo "F"-⁶⁰ is designed for free-hanging exposed applications. This is a standard model with an addition factory fitted architectural frame enhancement kit that can be finished in white to match the Halo beam, or provided as a different color to make a feature of the extruded aluminum outer frame.

Introduction

In addition to the flexibility offered by a modular designed small unit, Halo has been designed to deliver the most comfortable environment at any given air volume. Traditional active chilled beams with a ¹-way or ²-way throw have the potential to throw air at high velocities over long distances, however this may result in low comfort levels – particularly where the air streams from adjacent beams meet and fall downwards into the occupied zone or where beams are located close to walls or partitions.

Beams with a ⁴-way throw help to alleviate this problem, however the FTF Group's Halo beam takes the concept to the next level with its "true" ^{360°} diffusion pattern.

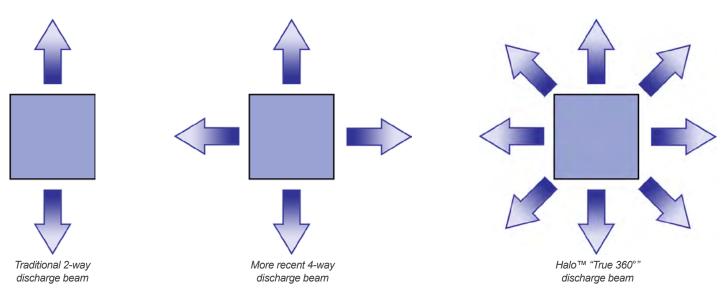
The substantially shorter air discharge throws (³⁵%) offered by Halo can enable more chilled beams to be positioned into a given room space for higher total heat gains to be offset whilst still avoiding drafts and providing a comfortable environment, compliant to BS EN ISO ⁷⁷³⁰/ASHRAE ⁵⁵.



Fig 2. Halo™ Active Chilled Beam 4ft x 4ft Module.

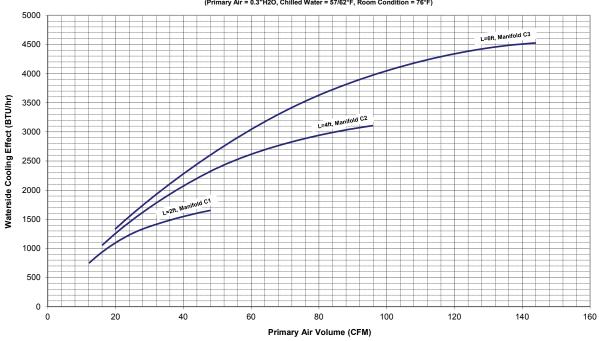


Fig 3. Halo™ Active Chilled Beam 4ft x 2ft Module fitted with architectural frame enhancement kit.



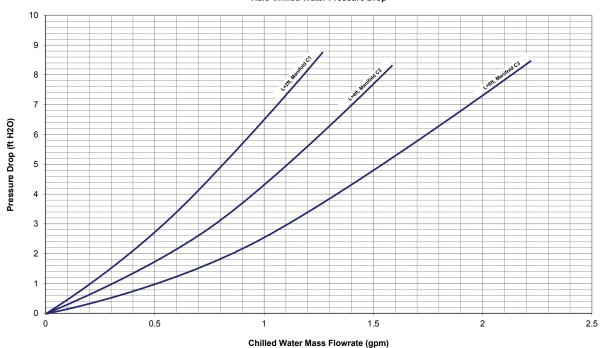
Halo distributes air in a 360° pattern for shorter air throws and optimum comfort.

Cooling Performance



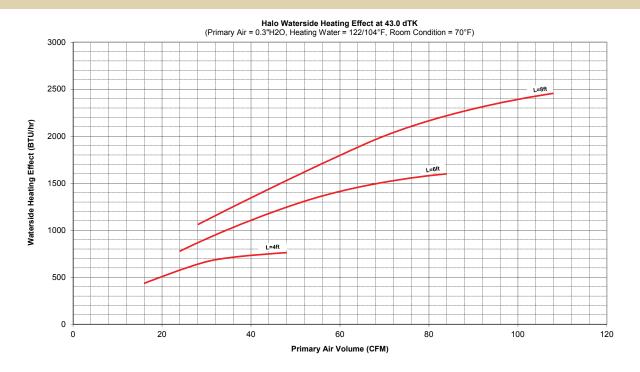
Halo Waterside Cooling Effect at 16.5 dTK (Primary Air = 0.3"H2O, Chilled Water = 57/62°F, Room Condition = 76°F)

Pressure Drop

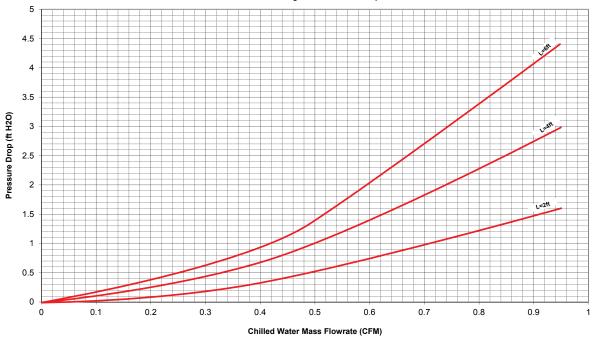


Halo Chilled Water Pressure Drop

Heating Performance



Pressure Drop



Halo Heating Water Pressure Drop

Cooling Selection Tables

Cooling at 0.24 Nozzle Pressure

Nozzle	Pressure H2O						Water										
	Halo		∆tK -	12.5°F			∆tK	- ¹⁴ . ⁶⁰ F			∆tK	- ¹⁶ . ^{6°} F			∆tK	- ¹⁸ . ^{6°} F	-
Q (CFM)	L (ft)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)
10	2.0	362	0_145	C1	0.6	466	0_186	C1	0.8	574	0,229	C1	1.0	684	0.273	C1	1,3
	2.0	666	0,266	C1	1.2	831	0.332	C1	1.7	988	0,394	C1	2.1	1136	0.454	C1	2.6
20	4.0	984	0.393	C1	5.8	1174	0.469	C1	7.6	1357	0.542	C1	9.6	1306	0.522	C ²	1.7
	6.0	937	0.374	C ²	1.8	1185	0.473	C ²	2.4	1435	0.573	C ²	3.1	1681	0.671	C ²	3.8
	2.0	866	0.346	C1	1.8	1064	0 425	C1	2.4	1247	0,498	C1	3.0	1424	0,569	C1	3.8
30	4.0	1287	0.514	C1	8.8	1524	0,609	C1	11.5	1533	0.612	C ²	2 0	1796	0.717	C ²	2.5
	6.0	1308	0.522	C ²	2.7	1621	0.647	C ²	3.6	1922	0.767	C ²	4.6	2208	0.882	C ²	5 <u>.</u> 6
	2.0	963	0.385	C1	2.1	1173	0.468	C1	2.8	1367	0.546	C1	3.5	1563	0,624	C1	4.3
40	4.0	1494	0,597	C1	11,1	1526	0,609	C ²	2.0	1829	0.731	C ²	2.6	2117	0,846	C ²	3 <u>.</u> 2
	6.0	1583	0,632	C ²	3.5	1930	0.771	C ²	4.6	2259	0,902	C²	5.8	2573	1 027	C ²	7.2
50	4.0	1427	0,570	C²	1.9	1782	0 712	C ²	2.5	2118	0.846	C²	3.2	2434	0.972	C ²	4.0
	6.0	1886	0 753	C ²	4.5	2265	0,905	C ²	5.9	2625	1 048	C ²	7.4	2989	1,194	C ²	9 <u>1</u>
60	4.0	1592	0,636	C ²	2.2	1979	0,790	C²	2.9	2338	0,934	C²	3.8	2677	1.069	C ²	4.7
	6.0	2142	0,856	C ²	5.4	2549	1 018	C ²	7.1	2962	1 179	C ²	8.9	3401	1,358	C ²	11_0
70	4.0	1715	0.685	C ²	2.4	2121	0.847	C ²	3.2	2494	0,996	C ²	4.2	2851	1 138	C ²	5.2
	6.0	2356	0.941	C ²	6.2	2795	1 116	C²	8,2	3260	1,302	C ²	10,4	3485	1 392	C3	4.2

Flow-adjusted waterside cooling effect table. Cooling circuit $\Delta t = 5^{\circ}F$ (Water in-out), nozzle pressure of $^{\circ}.^{24}$ inH₂O, $^{1} \times Q^{5^{\circ}}$ air connection.

Cooling at 0.32 Nozzle Pressure

Nozzle	Pressure H ₂ O								Wa	ater							
	Halo		∆tK -	12.5°F			∆tK	- ¹⁴ , ^{6°} F			ΔtK	- ¹⁶ . ^{6°} F			∆tK	- ¹⁸ . ^{6°} F	
Q (CFM)	L (ft)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)
10	2.0	440	0_176	C1	0.8	662	0.224	C1	1.0	685	0.274	C1	1.3	807	0.322	C1	1,6
	2.0	783	0.313	C1	1.5	962	0.384	C1	2.0	1130	0_451	C1	2.6	1289	0,515	C1	3.2
20	4.0	1129	0_451	C1	7.1	1333	0.532	C1	93	1543	0_616	C1	11.7	1543	0.616	C ²	2.1
	6.0	1147	0_458	C ²	2.3	1432	0.572	C ²	3.1	1711	0_683	C ²	3.9	1978	0,790	C ²	4.8
	2.0	1039	0_415	C1	2.3	1246	0_497	C1	3.0	1445	0.577	C1	3.8	1659	0.663	C1	4.7
30	4.0	1453	0.580	C1	10.7	1483	0,692	C ²	2.0	1780	0.711	C ²	2.5	2063	0.824	C ²	3.1
	6.0	1545	0_617	C ²	3.4	1885	0.753	C ²	4.4	2207	0.881	C ²	5.6	2512	1,003	C ²	6,9
	2.0	1184	0,473	C1	2,8	1404	0_561	C1	3.7	1635	0.653	C1	4.6	1922	0_767	C1	5.7
40	4.0	1432	0.572	C ²	1.9	1778	0_710	C ²	2.5	2105	0.841	C ²	3.2	2412	0.963	C ²	4.0
	6.0	1835	0.733	C ²	4.3	2207	0.881	C ²	5.6	2557	1.021	C ²	7.1	2905	1_160	C ²	8.7
	2.0	1285	0,613	C1	3.2	1519	0_607	C1	4.2	1788	0.714	C1	5.2	2160	0,863	C1	6.4
60	4.0	1710	0_683	C ²	2.4	2091	0.835	C ²	3.2	2445	0 976	C ²	4.0	2786	1,113	C ²	5 <u>.</u> 0
	6.0	2169	0,862	C ²	5.5	2564	1.024	C ²	7.2	2966	1 184	C ²	9.0	3416	1 364	C3	11.0
60	4.0	1943	0,776	C1	2.8	2347	0_937	C ²	3.8	2726	1 089	C ²	4.8	3113	1 243	C ²	5,9
	6,0	2443	0,976	C ²	6.6	2890	1 154	C ²	8.7	3382	1,351	C ²	10,9	3615	1 444	C3	4.4
70	4.0	2134	0,852	C ²	3,3	2561	1_019	C ²	4.3	2956	1 180	C ²	5.5	3400	1 358	C ²	6.7
	6.0	2695	1.076	C ²	7.7	3204	1,280	C ²	10_1	3496	1 396	C3	4.2	3968	1,693	C3	5.2

Flow-adjusted waterside cooling effect table. Cooling circuit $\Delta t = {}^{5\circ}F$ (Water in-out), nozzle pressure of ${}^{0.32}$ in H₂O, ${}^{1} \times Q^{5^{\circ}}$ air connection.

Cooling at 0.4 Nozzle Pressure

Nozzle	Pressure H ₂ O								Wa	ıter							
	Halo		∆tK -	^{12,6°} F			∆tK	- ¹⁴ . ⁵⁰ F			∆tK	- ¹⁶ . ^{6°} F			∆tK	- ¹⁸ .5°F	-
Q (CFM)	L (ft)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)	P (btu/h)	P (gpm)	Manifold	P (ft H ₂ O)
10	2.0	524	0,209	C1	0.9	664	0,265	C1	1.2	803	0.321	C1	16	936	0.374	C1	2.0
20	2.0	872	0.348	C1	1.8	1060	0.423	C1	2.4	1234	0_493	C1	3.0	1403	0.560	C1	3.7
	4.0	1277	0,610	C1	8.7	1508	0,602	C1	11.3	1525	0,609	C ²	2.0	1787	0.714	C ²	2.5
	2.0	1124	0_449	C1	2.6	1337	0.534	C1	3.4	1552	0_620	C1	4.3	1806	0.721	C1	5.3
30	4.0	1344	0.537	C ²	1.7	1676	0,669	C ²	2.3	1992	0.795	C ²	2.9	2288	0,914	C ²	3.6
	6.0	1785	0.713	C ²	4 1	2152	0.859	C ²	5.4	2495	0_997	C ²	6.8	2832	1 131	C ²	8.4
	2.0	1263	0,505	C1	31	1497	0.598	C1	4.1	1764	0 705	C1	5 1	2141	0.855	C1	6.3
40	4.0	1602	0.640	C²	2.2	1966	0.785	C ²	2.9	2306	0_921	C ²	3.7	2627	1 049	C ²	4.5
	6,0	2067	0,825	C²	5 1	2460	0,982	C ²	6.7	2840	1,134	C ²	8.4	3249	1_297	C ²	10_3
	2.0	1365	0.545	C1	3.5	1625	0_649	C1	4.6	1959	0.782	C1	5.8	2512	1_003	C1	7.2
60	4.0	1879	0_750	C ²	2.7	2271	0_907	C ²	3.6	2638	1 053	C ²	4.6	3005	1,200	C ²	5.6
	6.0	2379	0,950	C ²	6.3	2811	1_122	C ²	8.3	3273	1_307	C ²	10_4	3520	1_405	C3	4.2
	2.0	1389	0.555	C1	3.6	1667	0.662	C1	4.7	2015	0.805	C1	6.0	2639	1 054	C1	7.5
60	4.0	2106	0.841	C ²	3,2	2520	1,006	C ²	4.2	2920	1,166	C ²	5.4	3358	1,341	C ²	6 6
	6,0	2665	1,060	C²	7.6	3149	1,258	C ²	9,9	3448	1_377	C3	4.1	3930	1,670	C3	5.0
70	4.0	2289	0.914	C ²	3.6	2721	1 087	C ²	4.8	3162	1_263	C ²	6.1	3693	1 475	C ²	7.4
	6,0	2908	1_161	C ²	8.7	3499	1,397	C ²	11,4	4361	1 742	C3	4.7	4304	1,719	C3	5.8

Flow-adjusted waterside cooling effect table. Cooling circuit $\Delta t = {}^{5\circ}F$ (Water in-out), nozzle pressure of ${}^{\circ}.{}^{4}$ inH₂O, ${}^{1} \times \mathcal{O}^{5^{\circ}}$ air connection.

Heating Selection Tables

Heating at 0.24 Nozzle Pressure

Nozzle	Pressure H ₂ O						Wa	ater					
	Halo		∆tK - ^{36°} F			∆tK - ⁴⁵°F			∆tK - ^{54°} F			∆tK - ^{63°} F	
Q (CFM)	L (ft)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)
10	2.0	348	0,190	0,1	425	0_190	0,1	502	0_190	0,1	579	0,190	0,1
	2.0	504	0,190	0,1	615	0_190	0,1	725	0_190	0,1	835	0,190	0,1
20	4.0	647	0,190	0,2	789	0_190	0.2	962	0,214	0.2	1170	0,260	0,3
	e o	756	0,190	0,3	963	0,212	0.3	1200	0_267	0,5	1460	0.325	0.7
	2.0	605	0,190	0,1	738	0_190	0,1	877	0_195	0,1	1056	0.237	0,1
30	4.0	797	0,190	0,2	1024	0.228	0.2	1291	0.287	0,4	1570	0,350	0,5
	e`o	950	0,211	0,3	1261	0,281	0,5	1590	0.354	0,8	1934	0,430	11
	2.0	650	0,190	0,1	792	0_190	0_1	968	0,215	0.1	1177	0,262	0,2
40	4.0	912	0,203	0.2	1210	0,269	0.3	1526	0.340	0.5	1856	0,413	0.7
	e`o	1127	0,251	0.4	1497	0.333	0.7	1887	0,420	1,1	2292	0,510	1.5
59	4.0	1061	0,236	0.3	1409	0,314	0.4	1777	0,396	0.7	2160	0,481	0 <u>.</u> 9
	e o	1334	0,297	o'e	1773	0,395	1.0	2232	0_497	1,4	2705	0,602	2.0
60	4.0	1179	0,262	0.3	1566	0.349	0.5	1974	0_439	0.8	2396	0.533	1,1
3	e o	1523	0,339	0.7	2022	0,450	1.2	2541	0,666	1.8	3072	0,684	2.5
70	4.0	1262	0,281	0.4	1677	0.373	0.6	2112	0,470	0.9	2962	0,670	1,2
	e`o	1690	0,376	0 <u>.</u> 9	2241	0,499	1.4	2811	0,626	2 1	3390	0.755	3.0

Flow-adjust waterside heating effect table. Heating circuit $\Delta t = {}^{9\circ}F$ (Water in-out), nozzle pressure of ${}^{\circ,24}$ inH₂O, ${}^{1}x Q^{9\circ}$ air connection. For red values, the flow rate has been adjusted to the recommended minimum flow of ${}^{\circ,9}$ gpm.

Heating at 0.32 Nozzle Pressure

Nozzle	Pressure H ₂ O						Wa	iter					
	Halo		∆tK - ^{36°} F			∆tK - 45°F			∆tK - ^{54°} F			∆tK - ^{63°} F	
Q (CFM)	L (ft)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)
10	2.0	387	0,190	0,1	473	0_190	0,1	660	0_190	0_1	645	0,190	0,1
	2.0	657	0,190	0,1	682	0_190	0,1	802	0_190	0,1	983	0,212	0,1
20	4.0	715	0,190	0.2	881	0_196	0.2	1109	0_247	0,3	1350	0,300	0.4
	e`o	844	0,190	0,3	1114	0.248	0_4	1405	0.313	0,6	1709	0,380	0,9
	2.0	680	0,190	0,1	827	0_190	0,1	1031	0.229	0,1	1253	0.279	0,2
30	4.0	885	0_197	0,2	1173	0,261	0,3	1480	0_329	0_5	1800	0,401	0.7
	6 _. 0	1094	0,244	0.4	1453	0,323	0.7	1833	0_408	1.0	2227	0,495	1.4
	2.0	744	0_190	0,1	933	0,208	0_1	1175	0_262	0_2	1430	0,318	0,2
40	4.0	1048	0,233	0.3	1392	0,310	0_4	1755	0,391	0,6	2133	0 475	0,9
	e`o	1293	0,288	0.e	1718	0.382	0.9	2163	0.481	1.4	2623	0.584	1.9
	2.0	792	0,190	0,1	1017	0,226	0,1	1282	0,285	0,2	1560	0.347	0.3
50	4.0	1233	0,274	0,3	1638	0.364	0.6	2064	0,459	0,9	2504	0.557	1,2
	e o	1531	0,341	0.7	2032	0 452	1.2	2554	0,568	1.8	3087	0.687	2.5
60	4.0	1392	0,310	0.4	1849	0,411	0.7	2327	0.518	11	2818	0.627	1.5
	e o	1754	0,390	0 <u>.</u> 9	2325	0.517	1.5	2914	0.649	2,3	3511	0.781	3.2
70	4.0	1522	0,339	0,5	2021	0,450	0.8	2540	0,665	1.2	3071	0.683	1.7
.0	e`o	1958	0,436	11	2591	0.577	1.9	3240	0.721	2.7	3892	0,866	3.8

Flow-adjust waterside heating effect table. Heating circuit ∆t = ⁹°F (Water in-out), nozzle pressure of ^{0,32} inH₂O, ¹ x Ø⁹^r air connection. For red values, the flow rate has been adjusted to the recommended minimum flow of ^{0,19} gpm.

Heating at 0.4 Nozzle Pressure

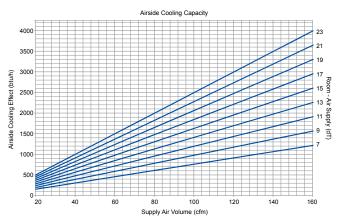
Nozzle	Pressure H ₂ O						Wa	iter					
	Halo		∆tK - ^{36°} F			∆tK - ^{45°} F			∆tK - ^{54°} F			∆tK - ^{63°} F	
Q (CFM)	L (ft)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)	P (btu/h)	P (gpm)	P (ft H ₂ O)
10	2.0	431	0,190	0,1	527	0_190	0,1	623	0_190	0,1	717	0_190	0,1
20	2.0	698	0_190	0,1	728	0_190	0_1	863	0_192	0_1	1048	0,233	0_1
	4.0	789	0,190	0,2	1012	0,225	0.2	1276	0.284	0.4	1553	0.346	0,6
	2.0	716	0,190	0,1	884	0_197	0,1	1113	0.248	0.2	1354	0,301	0,2
30	4.0	987	0,220	0,2	1310	0.291	0.4	1652	0.368	0,6	2009	0.447	0.8
	e.0	1256	0,280	0,5	1669	0.371	0.9	2103	0.468	1.3	2551	0.568	1.8
	2.0	782	0,190	0,1	1000	0.223	0_1	1261	0.281	0.2	1534	0.341	0.3
40	4.0	1152	0,256	0,3	1531	0.341	0.5	1930	0.429	0.8	2343	0.522	11
	e`o	1458	0,324	0.7	1936	0,431	11	2435	0.542	1.7	2947	0,656	2.3
	2.0	830	0,190	0,1	1088	0.242	0_1	1372	0,305	0,2	1670	0.372	0.3
50	4.0	1340	0,298	0.4	1780	0,396	0.7	2241	0,499	1.0	2715	0,604	1.4
	e.0	1699	0,378	0.9	2253	0.501	1.6	2826	0 629	2.2	3408	0.758	3.0
	2.0	841	0,190	0,1	1109	0.247	0.2	1398	0.311	0.2	1701	0,379	0.3
60	4.0	1501	0.334	0,5	1993	0.444	0.8	2506	0.558	1.2	3030	0.674	1.7
	e.0	1924	0,428	1,1	2546	0.567	1.8	3185	0 709	2.7	3828	0.852	3.7
70	4.0	1634	0.364	0,6	2168	0.483	0.9	2722	0,606	1.4	3285	0.731	1.9
	e`o	2130	0.474	1.3	2813	0,626	2 1	3509	0.781	3.2	4206	0,936	4.3

Flow-adjust waterside heating effect table. Heating circuit $\Delta t = {}^{9\circ}F$ (Water in-out), nozzle pressure of ${}^{0.4}$ inH₂O, ${}^{1} \times {}^{0^{9\circ}}$ air connection. For red values, the flow rate has been adjusted to the recommended minimum flow of ${}^{0.9}$ gpm.

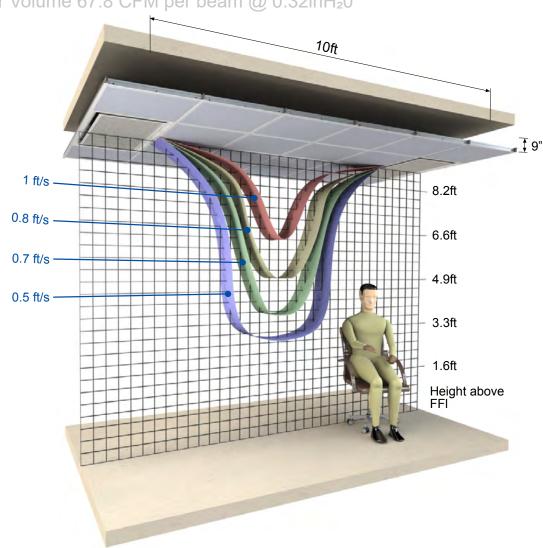
Air Cooling Effect

Cooling effect supplied in the ventilation air

- ¹. Start by calculating the required cooling effect that has to be supplied to the room in order to provide a certain temperature.
- ². Calculate any cooling effect that is provided by the ventilation air.
- ³. The remaining cooling effect has to be supplied by the beam.



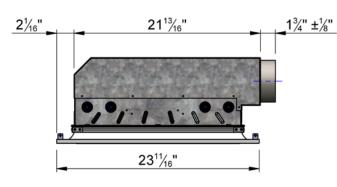
Air cooling effect as a function of airflow.

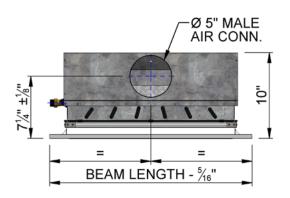


Scatter Diagram Fresh Air Volume 67.8 CFM per beam @ 0.32inH₂0

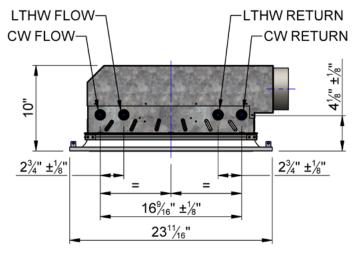
Product Dimensions

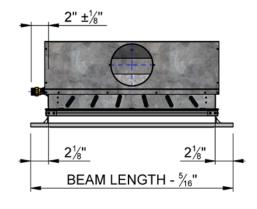
Air Connection



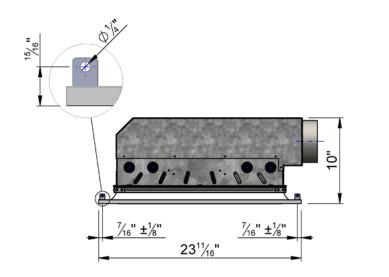


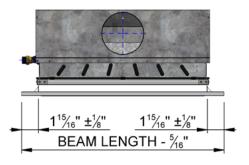
Water Connections



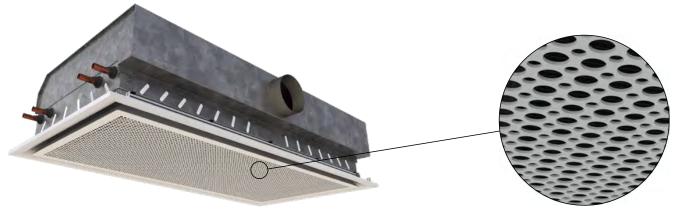


Mounting Details





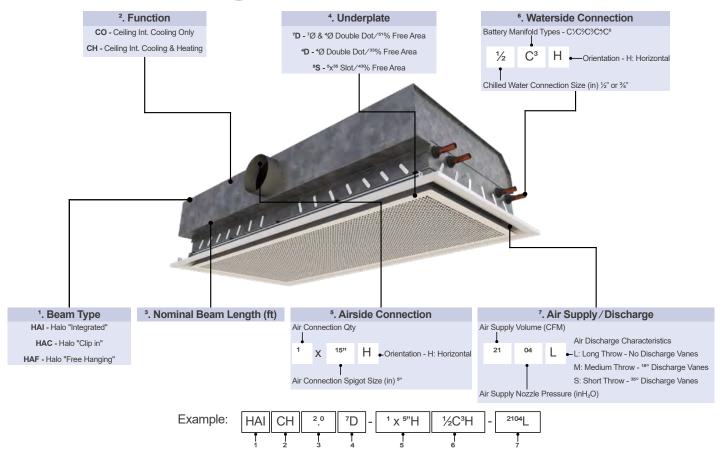
Perforation Pattern Options



Note: Other aesthetic options are available on request.

Double Dot Perforation 51% Free Area

Product Ordering Codes



Calculation Program

FTE GROUP®



Halo Active Beam Data	
Halo Type	Standard
Air Connection Orientation	Horizontal
Air Connection	1x5"
Product Length	6' ft
Manifold Type	C2
 Air Discharge Throw	L
Nozzle Static Pressure	0.4 <mark>" H2O</mark>
Fresh Air Volume	70 CFM
Underplate Perforation Type	51% DOT
Heating Function	Yes
Ceiling System	Lay In Grid

The FTF Group's calculation program for Halo is extremely user friendly.

Simply select from the drop down menu the "Air Connection" configuration. Air volumes in excess of 84.8 CFM and up to 106 CFM should be 2 x 80" diameter.

"Manifold types" can be changed in the drop down menu for increased waterside cooling effect, however attention needs to be taken regarding resultant pressure drops (hydraulic resistance). If pressure drops need reducing, chose a higher numbered manifold (C^5 being the highest and C^2 being the lowest).

"Discharge Throw" can be S (short), M (medium) or L (long).

"Underplate Perorated" options can be found on page ¹¹.

Design Conditions	Cooling		Heating	
Flow Water Temperature	57.0	°F	122.0	°F
Return Water Temperature	63.0	°F	113.0	°F
 Air Supply Temperature	61.0	°F	60.0	°F
Average Room Condition	75.0	°F	69.0	°F
"Air On" Thermal Gradient	1.2	°F		
Room Relative Humidity	50.0	%		

Complete your project data in the "Design Conditions" section. Please note that the "Air On" Thermal Gradient should not be used in normal instances

	Performance Data	Cooling		Heatin	g
	Air On - Mean Water dT	16.20	°F	48.50	°F
	Waterside Performance	3493	BTU/Hr	3083	BTU/Hr
_	Waterside Mass Flowrate	1.162	gpm	0.686	gpm
	Waterside Pressure Drop	8.8	ft H2O	2.4	ft H2O
	Airside Performance	1083	BTU/Hr	-696	BTU/Hr
	Total Sensible Performance	4756	BTU/Hr	2387	BTU/Hr
	Sound Effect Lw	<35	dB(A)		

"Performance Data" will then be automatically be calculated. Likewise "Dimensional Date" will be also automatically calculated.

Finally, the "Design Check" should read "Ok" in green, or detail some warnings in red.

Calculation program's for Halo are available upon request.

Contact our technical department or complete an application request form www.ftfgroup.us from the relevant link on our home page.

et Ref.										
lalo Active Beam Data		-	-							
Halo Type		Star	ndard							
ir Connection Orientation		Horiz	ontal						_	
Air Connection			1x5"			-				
Product Length			6'				11			
Manifold Type			C2						11,	
Air Discharge Throw			L		Γ					-
Nozzle Static Pressure			0.4	H2O				10.000		
Fresh Air Supply Volume			70	CFM						
Underplate Perforation Type		51%	DOT							
Heating Function			Yes							
Ceiling System		Lay Ir	Grid							
Design Conditions	Coolir	ng	Heatin	g		Dimensional Data				
Flow Water Temperature	57.0	°F	122.0	°F		Width x Depth	2ft x	10"	mm	
Return Water Temperature	63.0	°F	113.0	°F		Overall Length	6	•	ft	
Air Supply Temperature	61.0	°F	60.0	°F	-	Water Volume	_ 1.	0	gal	
Average Room Condition	75.0	°F	69.0	°F		Dry Weight	10	5.4	lb	
"Air On" Thermal Gradient	1.2	°F				CW Connection	1/2"	NPT		
Room Relative Humidity	50.0	%				LTHW Connection	1/2"	NPT		
Performance Data	Cooling	-	Heatin	a		Design Check (Warnin	ngs)			
Air On - Mean Water dT	16.20		48.5			Air Discharge OK				
Vaterside Performance	3493	BTU/Hr	3083	BTU/Hr		Supply Air OK				
Vater Mass Flowrate	1.162	gpm	0.686	gpm	I	Cooling Circuit OK				
Vaterside Pressure Drop		ft H2O	2.5	ft H2O	-		-	-	-	
Airside Performance	1083	BTU/Hr	-696	BTU/Hr		Heating Circuit OK				
otal Sensible Performance	4576	BTU/Hr	2387	BTU/Hr		Turn Down Vol @ 40 P	а		5.6	CFM
Sound Effect LW	< 35				L	Calculated Dew Point		5	5.1	°F

Notes: 1) Performance calculations are based upon normal clean potable water; it is the system engineer's responsibility to allow for any reduction in cooling or heating performance due to additives that may reduce the water systems heat transfer coefficient.

2) Pressure drop calculations are based upon ASHRAE guides using clean potable water and exclude any additional losses associated with entry / exit losses, pipe fouling or changes in water quality; it is the system engineer's responsibility to use good engineering practice.

Air discharge throw guidance based on beams on 10 foot centres for alternative layouts contact FTF Technical Dept for throw settings

Project Specific Testing Facility

The FTF Group have ³ number state-of-the-art Climatic Testing Laboratories at one if its subsidiary companies predominantly situated at the prestigious Pride Park. Each laboratory has internal dimensions of ²⁰.⁷ft x ¹⁸.⁷ft x ¹⁰.⁸ft high and includes a thermal wall so that both core and perimeter zones can be modeled. The test facilities are fixed in overall size and construction therefore simulation of a buildings specific thermal mass cannot be completed, it should, however be noted that a specific project can be simulated more accurately by recessing the floor and reducing the height at necessary.

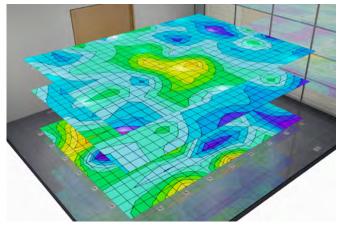
Project Specific Testing

Project specific mock-up testing is a valuable tool which allows the Client to fully asses the proposed system and determine the resulting indoor quality and comfort conditions; the physical modeling is achieved by installing a full scale representation of a building zone complete with internal & external heat gains (Lighting, Small Power, Occupancy & Solar Gains).

The installed mock-up enables the client to verify the following:

- Product performance under project specific conditions.
- Spatial air temperature distribution.
- Spatial air velocities.
- Experience thermal comfort.
- Project specific aesthetics.
- Experience lighting levels (where relevant).
- Investigate the specific design and allow the system to be enhanced.







The project-specific installation and test is normally conducted to verify:





Photometric Testing Facility

The FTF Group's technical facility at Pride Park, Derby also has two Photometric test laboratories which are used to evaluate the performance of luminaires. To measure the performance, it is necessary to obtain values of light intensity distribution from the luminaire. These light intensity distributions are used to mathematically model the lighting distribution envelope of a particular luminaire. This distribution along with the luminaires efficacy allows for the generation of a digital distribution that is the basis of the usual industry standard electronic file format. In order to assess the efficacy of the luminaire it is a requirement to compare the performance of the luminaire against either a calibrated light source for absolute output or against the "bare" light source for a relative performance ratio.

The industry uses both methods. Generally absolute lumen outputs are used for solid state lighting sources and relative lighting output ratios (LOR) are used for the more traditional sources. Where the LOR method is chosen then published Lamp manufacturer's data is used to calculate actual lighting levels in a design.

The intensity distribution is obtained by the use of a Goniophotometer to measure the intensity of light emitted from the surface of the fitting at pre-determined angles. The light intensity is measured using either a photometer with a corrective spectral response filter to match the CIE standard observer curves or our spectrometer for LED sources.

Luminaire outputs are measured using out integrating sphere for small luminaires or out large integrator room for large fittings and Multiservice Chilled Beam. For both methods we can use traceable calibrated radiant flux standards for absolute comparisons.

All tests use appropriate equipment to measure and control the characteristics of the luminaire and include air temperature measurements, luminaire supply voltage, luminaire current and power. Thermal characteristics of luminaire components can be recorded during the testing process as required.

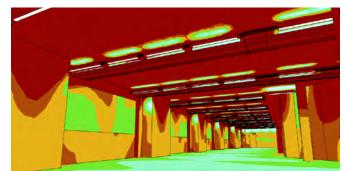
A full test report is compiled and supplied in "locked" PDF format. Data is collected and correlated using applicable software and is presented electronically to suit, usually in Eulumdat, CIBSE TM¹⁴ or IESN standard file format.

The FTF Groups technically facility also conducts photometric tests in accordance with CIE ¹²⁷:²⁰⁰⁷ and BS EN ¹³⁰³²-¹ and sound engineering practice as applicable. During the course of these tests suitable temperature measurements of parts of LEDs can be recorded. These recorded and plotted temperature distributions can be used to provide feedback and help optimize the light output of solid state light source based luminaires which are often found to be sensitive to junction temperature.











Acoustic Testing Facility

The Acoustic Test Room at the FTF Groups Technical Facility is a hemi-anechoic chamber which utilises sound absorbing acoustic foam material in the shape of wedges to provide an echo free zone for acoustic measurement; the height of the acoustic foam wedges has a direct relationship with the maximum absorption frequency, hence the FTF Group has the wedges specifically designed to optimise the sound absorption at the peak frequency normally found with our Active Chilled Beam products.

The use of acoustic absorbing material within the test room provides the simulation of a quiet open space without "reflections" which helps to ensure sound measurements from the sound source are accurate, in addition the acoustic material also helps reduce external noise entering the test room meaning that relatively low noise levels of sound can be accurately measured.

The acoustic facilities allow the FTF Group to provide express in-house sound evaluation so that all products, even project specific designs can be assessed and optimised.

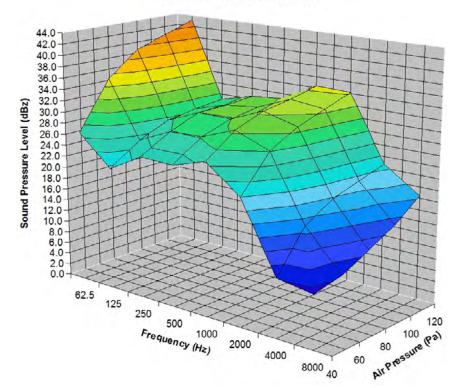
To ensure accuracy the FTF Group only use Class ¹ measurement equipment which allows sound level measurements to be taken at ¹¹ different ¹/₃ octave bands between ¹⁶ Hz to ¹⁶ kHz, with A, C and Z (un-weighted) simultaneous weightings.

In addition to the above, the FTF Group also send their new products for specialist third party Acoustic Testing. The results of which are very close and within measurement tolerances to that of FTF Groups in-house measurement of sound.





Unweighted Sound Pressure Level





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